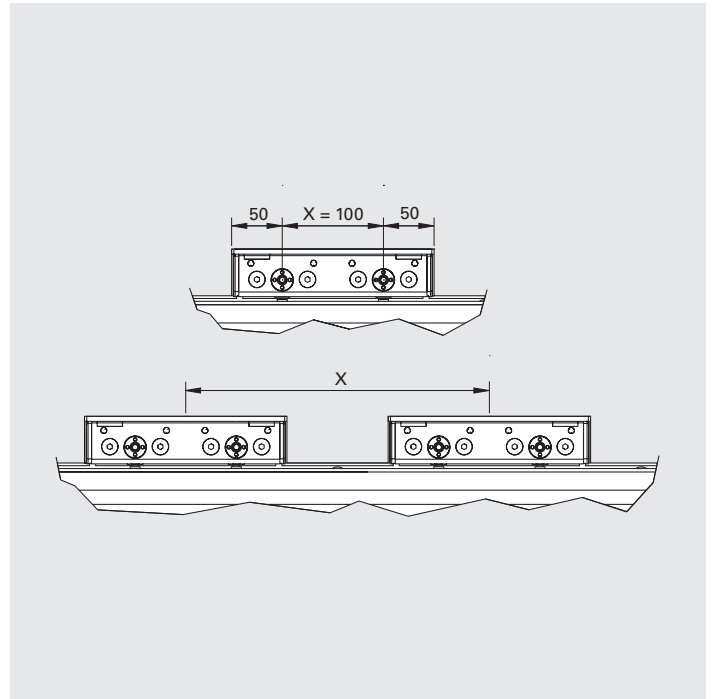
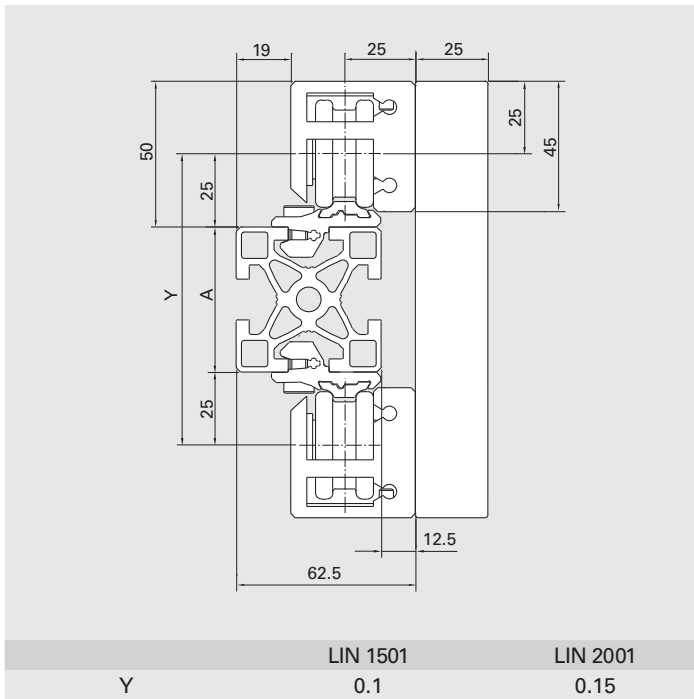


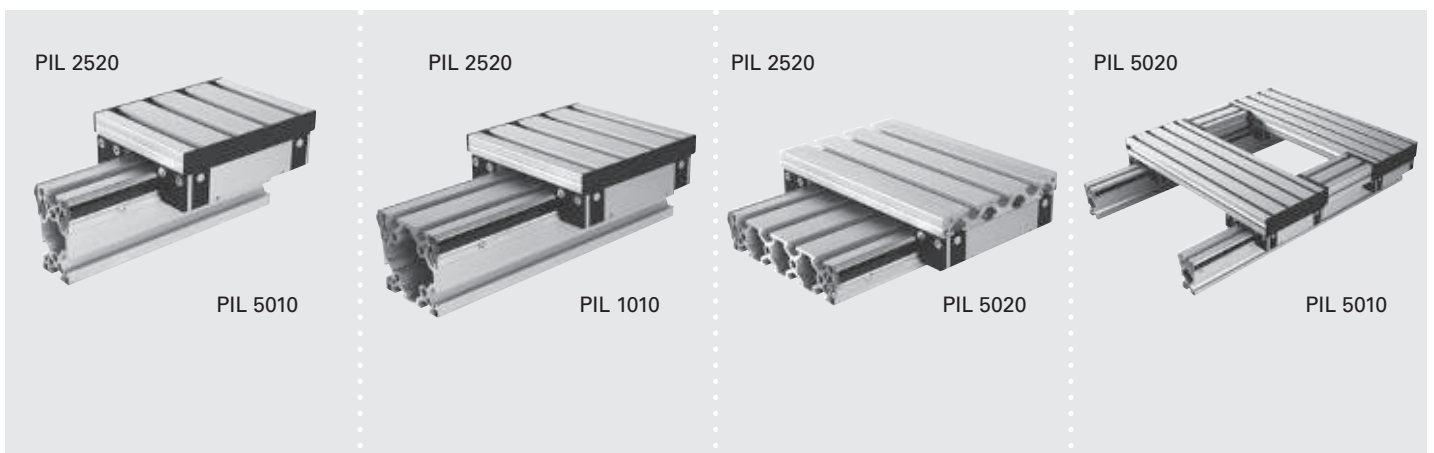
## Linear Motion System Description Key

$F_{(x,y,z) \text{ max.}}$	[N]	maximum rated external static load
$F_{(x,y,z)}$	[N]	actual external static load
$F_R$	[N]	maximum rated load / roller - radial (max. 1500 N)
$F_A$	[N]	Maximum rated load / roller - axial (max. 750 N)
$M_{(x,y,z) \text{ max.}}$	[Nm]	maximum rated static torque load
$a_{\text{max.}}$	[m/s <sup>2</sup> ]	maximum rated acceleration
X	[m]	distance between rollers in direction of motion
Y	[m]	distance between rollers perpendicular to direction of motion
A	[m]	extrusion width between rollers
g	[m/s <sup>2</sup> ]	gravitational acceleration (approx. 9.81 m/s <sup>2</sup> )
$m_1$	[kg]	mass of carriage and lever
$m_2$	[kg]	mass of mounted parts
$F_{a \text{ req.}}$	[N]	required drive power
$M_{d \text{ hor req.}}$	[Nm]	required drive torque in horizontal direction
$M_{d \text{ ver req.}}$	[Nm]	required drive torque in vertical direction
$L_1$	[m]	distance of center of gravity of carriage plate and lever
L2	[m]	distance of center of gravity of mounted parts and/or distance of forces $F(x,y,z)$

## Linear Motion System Dimensions

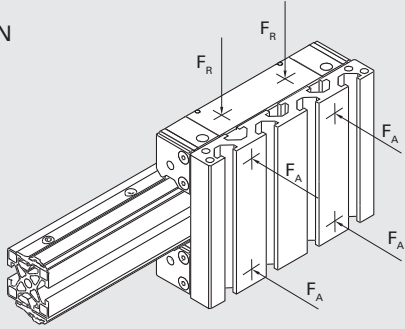


## Linear Motion System Carriage Combination Examples



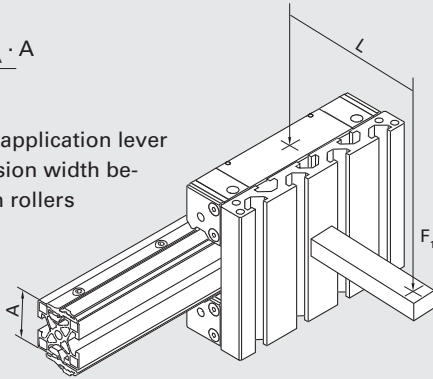
# Linear Motion System Load Capacity

$F_R < 1500 \text{ N}$   
 $F_A < 750 \text{ N}$



$$F_1 \leq \frac{2 \cdot F_A \cdot A}{L}$$

L = Force application lever  
 A = extrusion width between rollers



### Technical Data

Acceleration:  $a_{max} < 15 \text{ m/s}^2$   
 Max. drive torque: 60 Nm  
 Forces:  $F_R$  [N] permissible load / roller - radial (max. 1500 N)  
 $F_A$  [N] permissible load / roller - axial (max. 750 N)

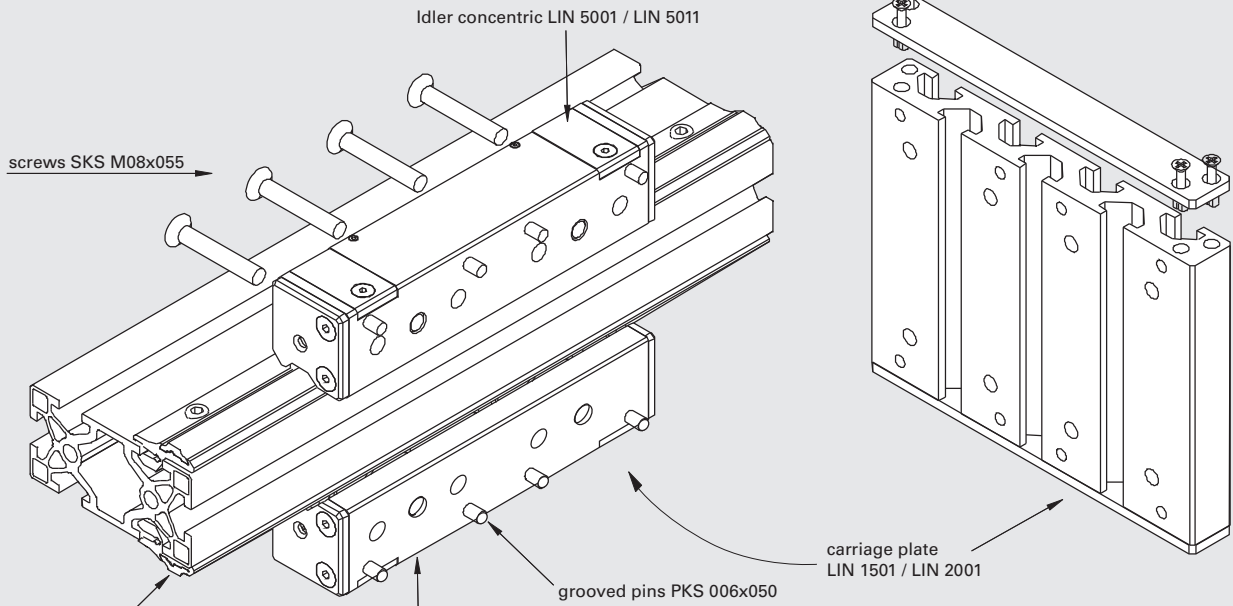
### Maintenance:

Lubrication: In order to ensure maximum service life of the guide-rail make sure that it is sufficiently lubricated.  
 Check the scraper and lubrication devices on a regular basis and re-lubricate as needed.

Load: Load data is valid for one type of load and one direction of load.

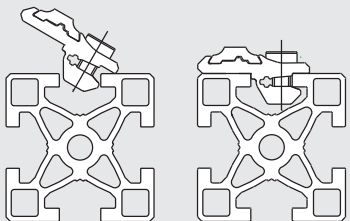
“Static design” refers to weight and operation forces. “Dynamic design” refers to weight and acceleration forces.

# Linear Motion System Assembly Tips



installation of guide rail  
 LIL 5000 SNN

Idler eccentric LIN 5003 / LIN 5013

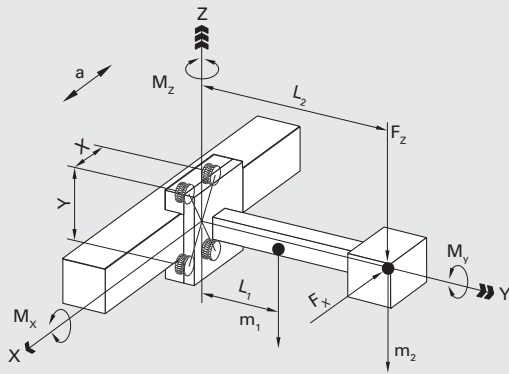


insert

press to right-hand side  
 and tighten

# Linear Motion System Calculations

## Application A (horizontal)



Static design:

$$M_{X \max.} = 2 \cdot F_A \cdot A$$

$$F_{Z \max.} = \frac{M_{X \max.} - g \cdot (m_1 \cdot L_1 + m_2 \cdot L_2)}{L_2}$$

$$M_{Z \max.} = 2 \cdot F_A \cdot X$$

$$F_{X \max.} = \frac{M_{Z \max.}}{L_2}$$

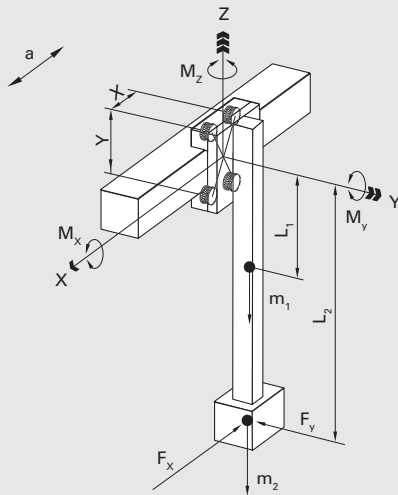
Dynamic design:

$$a_{\max.} = \frac{F_A \cdot X}{(m_1 \cdot L_1 + m_2 \cdot L_2) \cdot 4} - \frac{g \cdot X}{9 \cdot Y}$$

$$F_{a \text{ req.}} = (m_1 + m_2) \cdot a_{\max.} + 10N$$

$$M_{d \text{ hor req.}} = F_{a \text{ req.}} \cdot 0.035 \text{ m} \cdot 1.8$$

## Application B (horizontal)



Static design:

$$M_{X \max.} = 2 \cdot F_A \cdot A$$

$$F_{Y \max.} = \frac{M_{X \max.}}{L_2}$$

$$M_{Y \max.} = (F_R - \frac{(m_1 + m_2) \cdot g}{2}) \cdot (\sqrt{x^2 + y^2} - 0.036)$$

$$F_{X \max.} = \frac{M_{Y \max.}}{L_2}$$

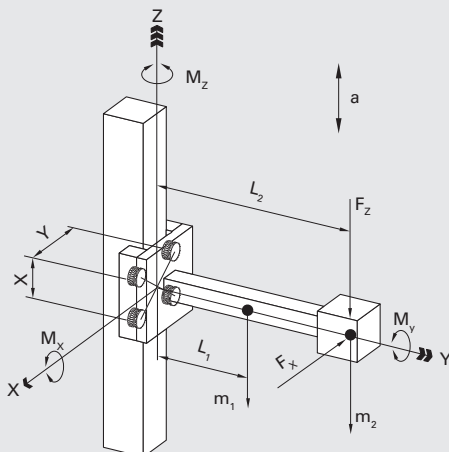
Dynamic design:

$$a_{\max.} = \frac{(F_R - \frac{(m_1 + m_2) \cdot g}{2}) \cdot (\sqrt{x^2 + y^2} - 0.036)}{(m_1 \cdot L_1 + m_2 \cdot L_2) \cdot 2}$$

$$F_{a \text{ req.}} = (m_1 + m_2) \cdot a_{\max.} + 10N$$

$$M_{d \text{ hor req.}} = F_{a \text{ req.}} \cdot 0.035 \text{ m} \cdot 1.8$$

## Application C (vertical)



Static design:

$$M_{X \max.} = 2 \cdot F_A \cdot X$$

$$F_{Z \max.} = \frac{M_{X \max.} - g \cdot (m_1 \cdot L_1 + m_2 \cdot L_2)}{L_2}$$

$$M_{Z \max.} = 2 \cdot F_A \cdot Y$$

$$F_{X \max.} = \frac{M_{Z \max.}}{L_2}$$

Dynamic design:

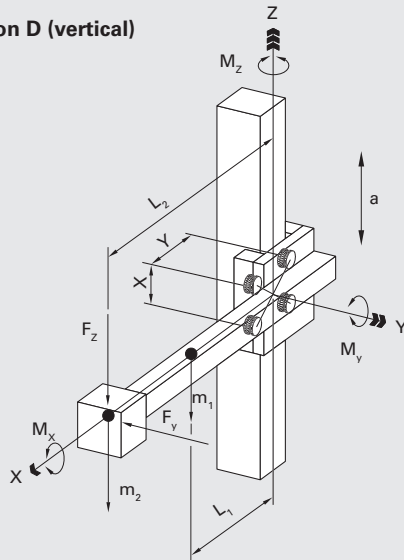
$$a_{\max.} = \frac{2 \cdot F_A \cdot X}{(m_1 \cdot L_1 + m_2 \cdot L_2) \cdot 2} - g$$

$$F_{a \text{ req.}} = (m_1 + m_2) \cdot (a_{\max.} + g) + 10N$$

$$M_{d \text{ ver req.}} = F_{a \text{ req.}} \cdot 0.035 \text{ m} \cdot 1.8$$

# Linear Motion System Calculations

## Application D (vertical)



Static design:

$$M_{Y \max.} = FR \cdot \sqrt{x^2 + y^2 - 0.036}$$

$$F_{Z \max.} = \frac{M_{Y \max.} - g \cdot (m_1 \cdot L_1 + m_2 \cdot L_2)}{L_2}$$

$$M_{Z \max.} = 2 \cdot F_A \cdot Y$$

$$F_{Y \max.} = \frac{M_{Z \max.}}{L_2}$$

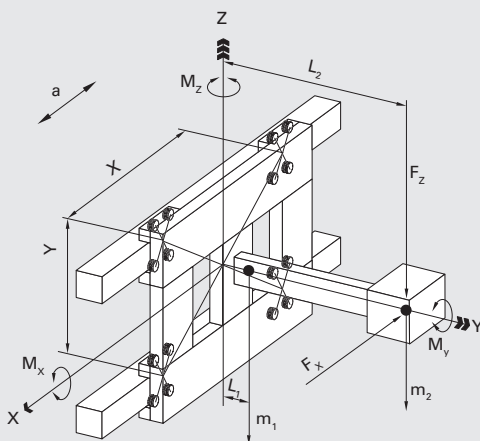
Dynamic design:

$$a_{\max.} = \frac{F_R \cdot (\sqrt{x^2 + y^2} - 0.036)}{(m_1 \cdot L_1 + m_2 \cdot L_2) \cdot 2} - g$$

$$F_{a \text{ req.}} = (m_1 + m_2) \cdot (a_{\max.} + g) + 10N$$

$$M_{d \text{ ver req.}} = F_{a \text{ req.}} \cdot 0.035 \text{ m} \cdot 1.8$$

## Application E (horizontal)



Static design:

$$M_{X \max.} = 8 \cdot F_A \cdot (y - 0.15)$$

$$F_{Z \max.} = \frac{M_{X \max.} - g \cdot (m_1 \cdot L_1 + m_2 \cdot L_2)}{L_2}$$

$$M_{Z \max.} = 4 \cdot F_A \cdot X$$

$$F_{X \max.} = \frac{M_{Z \max.}}{L_2}$$

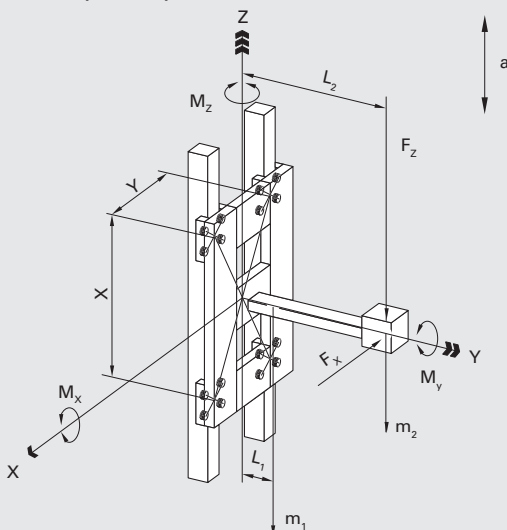
Dynamic design:

$$a_{\max.} = \frac{4 \cdot F_A \cdot X}{(m_1 \cdot L_1 + m_2 \cdot L_2) \cdot 2}$$

$$F_{a \text{ req.}} = (m_1 + m_2) \cdot a_{\max.} + 40N$$

$$M_{d \text{ hor req.}} = F_{a \text{ req.}} \cdot 0.035 \text{ m} \cdot 1.8$$

## Application F (vertical)



Static design:

$$M_{X \max.} = 4 \cdot F_A \cdot X$$

$$F_{Z \max.} = \frac{M_{X \max.} - g \cdot (m_1 \cdot L_1 + m_2 \cdot L_2)}{L_2}$$

$$M_{Z \max.} = 4 \cdot F_A \cdot Y$$

$$F_{X \max.} = \frac{M_{Z \max.}}{L_2}$$

Dynamic design:

$$a_{\max.} = \frac{4 \cdot F_A \cdot X}{(m_1 \cdot L_1 + m_2 \cdot L_2) \cdot 2} - g$$

$$F_{a \text{ req.}} = (m_1 + m_2) \cdot (a_{\max.} + g) + 40N$$

$$M_{d \text{ ver req.}} = F_{a \text{ req.}} \cdot 0.035 \text{ m} \cdot 1.8$$